



# Summary of the development activities in the EU for testing residential air conditioners

European Commission, DG Energy, Veerle Beelaerts

# Energy efficiency requirements for residential air conditioners



Residential air conditioners and heat pumps ( $\leq 12$  kW)



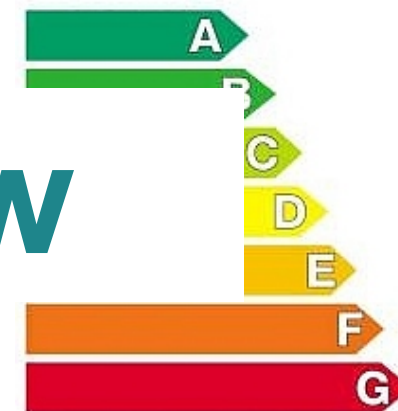
Ecodesign\*

Energy labelling\*\*

Energy efficiency



# Under review



\*Regulation (EU) No 206/2012

\*\*Regulation (EU) No 626/2011

# Basis for setting requirements – seasonal efficiency (SCOP and SEER)



EN 14825



- energy efficiency representative of the cooling and heating season (seasonal efficiency, i.e. SEER and SCOP)
- uses same basic principles as standard series ISO 16358 (ISO TC86 SC6)

# Concerns with the current testing method (1)

The current test method doesn't require manufacturers to take into account *thermal comfort*:

- In cooling mode **45% of the units do not dehumidify** (data from calculations from an EU manufacturer) -> dehumidification is necessary to ensure thermal comfort
- In heating mode the **temperature of the air** that blows out of the heat pump is **as low as 27°C and commonly lower than 32°C** -> the temperature of the air that blows out of the heat pump (supply air temperature) should not be below 32 °C (temperature of the skin) and probably closer to 40 °C to ensure thermal comfort.

In reality, when thermal comfort is not ensured, the end-user will change the set point. This will increase cooling/heating loads, and will lead to **lower real life performances**.

# Concerns with the current testing method (2)

The current test method:

- requires *manufacturers to give the settings of the unit* during test
- *bypasses the control*
- *locks the compressor* during test

This is a *worldwide practice*

However, the *performance of units in real life may differ* from the performances measured in standard test conditions

# Looking for solutions – ensuring thermal comfort

- **Heating:** set parameters (e.g. set values for air flow rate) such that the *temperature blowing* out of the heat pumps is *between 32°C and 40°C* (under discussion)
- **Cooling:** set parameters (e.g. max sensible heat ratio or limitation on the air flow rate) such that the:
  - *minimum sensible heat ratio is 70% at 35°C ambient temperature, and 95 % at 30°C* (proposal stakeholder), or alternatively
  - *minimum sensible heat ratio is 80 % at 35°C ambient temperature, and 85 % at 30°C* (US AHRI 1230 VRF)

# Looking for solutions – independent test method (1)

2 alternative methods have been proposed by stakeholders:

## 1) The compensation method

- Thermal load imposed to the machine, the unit has to maintain the set point, the compressor and outdoor fan are unlocked, real life control
- Same test conditioners as EN 14825

=> Round robin test is ongoing in cooling mode, for heating more tests might be needed

## 2) The dynamic method

- Same test method as the compensation method
- 21 times steps of 2.5 hours covering the whole load curve and outdoor air conditions

=> Further work is needed

# Proposal currently being discussed

Based on the above, a possible way forward that is currently being discussed:

- Tier 1 (1 year after entry into force, tentatively Mid-2023): improve the thermal comfort and set resource efficiency requirements
- Tier 2 (5 years after entry into force, tentatively Mid-2027): mandatory application of an independent method that doesn't fix the compressor and which fulfils the requirements for a method fit for regulatory purposes
- Review (7 years after entry into force, tentatively Mid 2029)



# Thank you



© European Union 2020

Unless otherwise noted the reuse of this presentation is authorised under the [CC BY 4.0](https://creativecommons.org/licenses/by/4.0/) license. For any use or reproduction of elements that are not owned by the EU, permission may need to be sought directly from the respective right holders.

Slide xx: [element concerned](#), source: [e.g. Fotolia.com](#); Slide xx: [element concerned](#), source: [e.g. iStock.com](#)



# Improving thermal comfort

## HEATING MODE

### POSSIBLE SCENARIOS: PRINCIPLES

1. Constant indoor air flow rate,  $T_{\text{supply}}$  equal to 32 °C at an outdoor air temperature equal to the bivalent temperature ( $T_{\text{biv}}$ );
2. Constant indoor air flow rate,  $T_{\text{supply}}$  equal to 40 °C at  $T_{\text{outdoor}}$  equal to  $T_{\text{biv}}$ ;
3. Variable air indoor flow rate,  $T_{\text{supply}}$  equal to 40 °C at  $T_{\text{outdoor}}$  equal to  $T_{\text{biv}}$  and  $T_{\text{supply}}$  equal to 32 °C at  $T_{\text{outdoor}}$  equal to 12°C (rating point D).
4. *New: Variable air indoor air flow rate in line with water based fan coil intermediate temperature regime (variable water temperature outlet) in EN14825 (40/45 @ -10 °C down to --/28 °C @ 12 °C), calculated here based on water outlet temperature with coil effectiveness of 0.85*

# Improving thermal comfort

## COOLING MODE

### POSSIBLE SCENARIOS: PRINCIPLES

1. Ensure minimum SHR of 70 % in A condition, and 95 % in B condition (Daikin proposal)
2. Ensure minimum SHR of 80 % in A condition, and 85 % in B condition (US AHRI 1230 VRF)